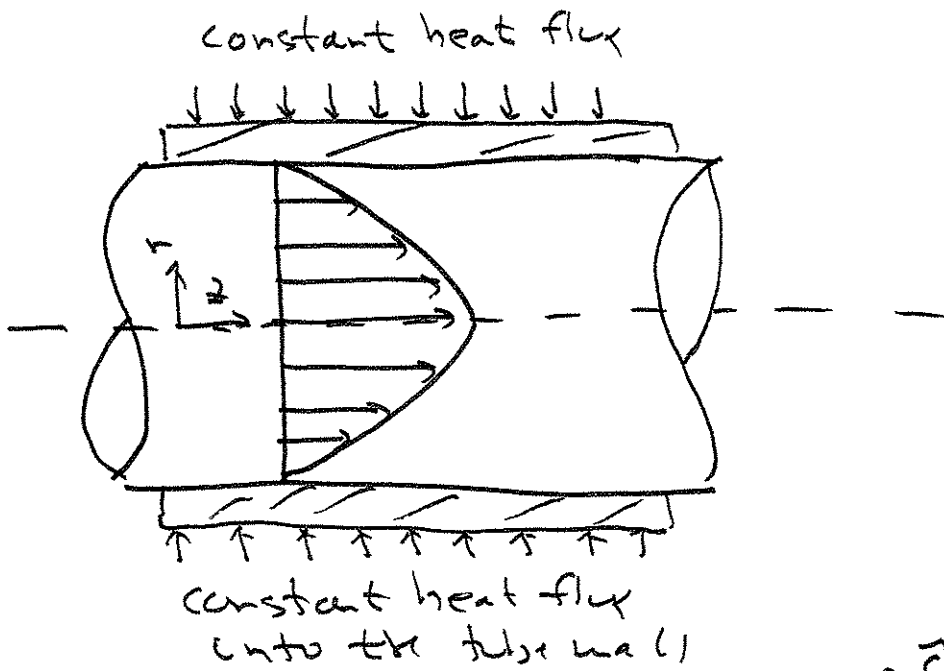


EXAMPLE 2

Nonisothermal Fluid Transport at a constant wall
Heat Flux with a laminar velocity profile



Laminar Flow Velocity Profile

$$u_z(r) = \frac{(P_0 - P_L)R^2}{4\mu L} \left[1 - \left(\frac{r}{R}\right)^2 \right]$$

$$= u_{z,\max} \left[1 - \left(\frac{r}{R}\right)^2 \right]$$

where: $P_0 = P_0 + \rho g L_0$; $P_L = P_L + \rho g L$

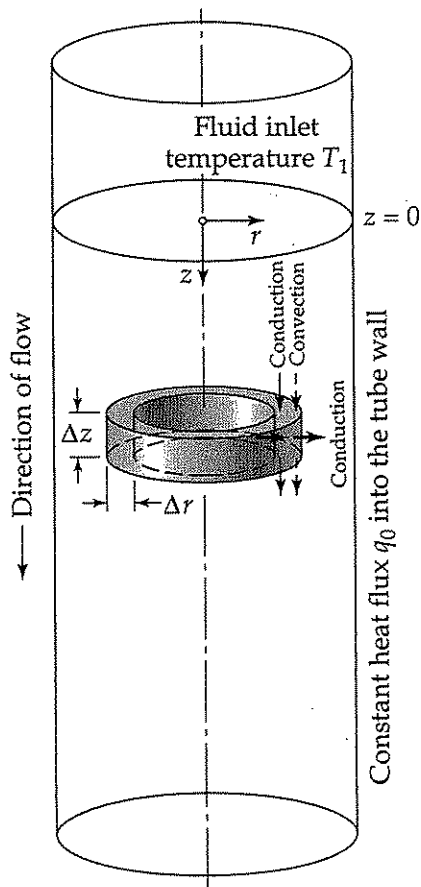
$$\rho \bar{c}_p u_{z,\max} \left[1 - \left(\frac{r}{R}\right)^2 \right] \frac{\partial T}{\partial z} = k \left[\frac{\partial^2 T}{\partial z^2} + \frac{1}{r} \frac{\partial}{\partial r} \left(r \frac{\partial T}{\partial r} \right) \right]$$

at $r = 0$, T is finite

at $r = R$, $k \frac{\partial T}{\partial r} = q_0$ (constant)

Nonisothermal Fluid Transport with a laminar velocity Profile-1

Heating of a fluid in laminar flow through a circular tube showing the differential control volume



Assumptions

1. Steady-state $\Rightarrow T \neq f(t)$; $T = T(r, z)$
2. Constant fluid properties ($\rho, \rho_p, \mu, k \dots$)
3. Constant radial heat flux is applied at the wall $q_r = -q_0$. If the wall is heated, then $q_0 = +$ and if chilled, then $q_0 = -$
4. Constant inlet temperature, i.e., $T = T_0$ at $z=0$ ($0 \leq r \leq R$)
5. The fluid velocity corresponds to a laminar flow profile $v_z(r) = v_{z, \max} \left[1 - \left(\frac{r}{R} \right)^2 \right]$
6. The variation in the velocity profile for $0 \leq r \leq R$ suggests that radial conduction of energy must be accounted for, in addition to axial conduction of energy
7. Fluid energy transport by convection occurs in the axial direction

Nonisothermal Fluid Transport with a Laminar Velocity Profile-2

- The average fluid velocity is obtained by dividing the fluid volumetric flow rate by the cross-sectional area

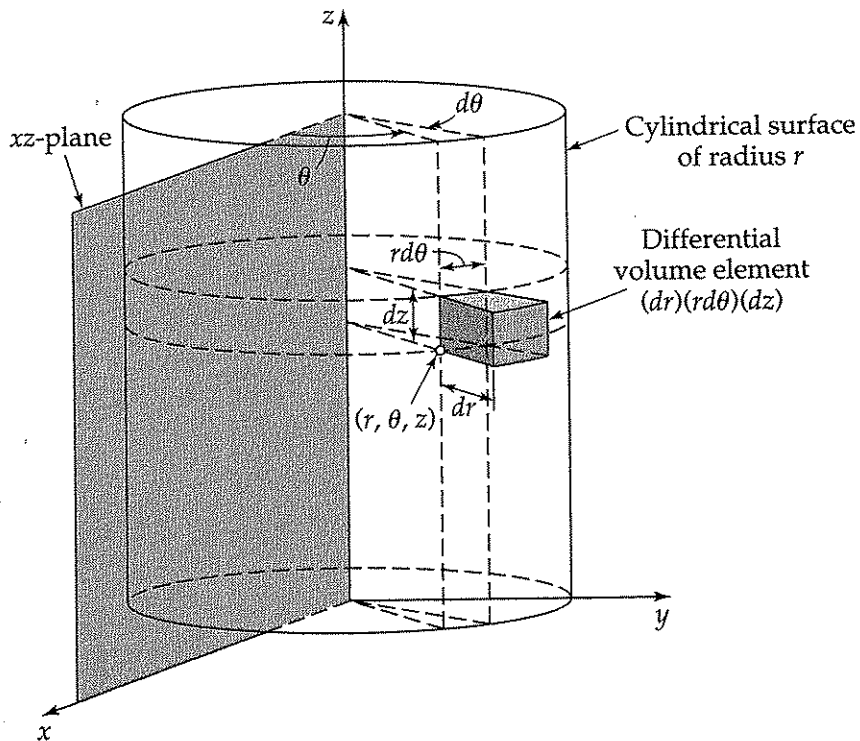


Fig. A.8-1. Differential volume element $r dr d\theta dz$ in cylindrical coordinates, and differential line elements dr , $r d\theta$, and dz . The differential surface elements are: $(r d\theta)(dz)$ perpendicular to the r direction (intermediate shading); $(dz)(dr)$ perpendicular to the θ direction (darkest shading); and $(dr)(r d\theta)$ perpendicular to the z direction (lightest shading).

$$v_z(r) = v_{z,max} \left[1 - \left(\frac{r}{R} \right)^2 \right]$$

$$dQ = v_z(r) dA$$

From the control volume, the differential area \perp to the z -direction is

$$dA = r dr d\theta$$

$$dQ = v_z(r) r dr d\theta$$



$$\langle v_z \rangle = \frac{\int dQ}{\int dA} = \frac{\int_0^{2\pi} \int_0^R v_z(r) r dr d\theta}{\int_0^{2\pi} \int_0^R r dr d\theta}$$

$$\text{Numerator} = Q = \int_0^{2\pi} \int_0^R v_{z,max} \left[1 - \left(\frac{r}{R} \right)^2 \right] r dr d\theta = v_{z,max} \int_0^{2\pi} \int_0^R \left(r - \frac{r^3}{R^2} \right) dr d\theta$$

$$= v_{z,max} \int_0^{2\pi} \left[\frac{r^2}{2} - \frac{r^4}{4R^2} \right]_0^R d\theta = v_{z,max} \left[\frac{R^2}{2} - \frac{R^4}{4R^2} \right] \int_0^{2\pi} d\theta = 2\pi v_{z,max} \frac{R^2}{4} = \frac{4\pi}{2} R^2 v_{z,max}$$

Nonisothermal Fluid Transport with a Laminar velocity Profile - 3

• Denominator = $A = \int_0^{2\pi} \int_0^R r dr d\theta = \int_0^{2\pi} \left[\frac{r^2}{2} \right]_0^R d\theta = \frac{2\pi R^2}{2} = \pi R^2$

$\langle v_z \rangle = \frac{Q}{A} = \frac{\text{numerator}}{\text{denominator}} = \frac{\frac{\pi}{2} R^2 v_{z,\max}}{\pi R^2} = \frac{v_{z,\max}}{2} \Rightarrow v_{z,\max} = 2\langle v_z \rangle$

• the laminar velocity profile is: $v_z(r) = v_{z,\max} \left[1 - \left(\frac{r}{R} \right)^2 \right]$

But: $v_{z,\max} = 2\langle v_z \rangle = 2v_0$ $= 2\langle v_z \rangle \left[1 - \left(\frac{r}{R} \right)^2 \right]$

Text notation \uparrow
Eq 1.21

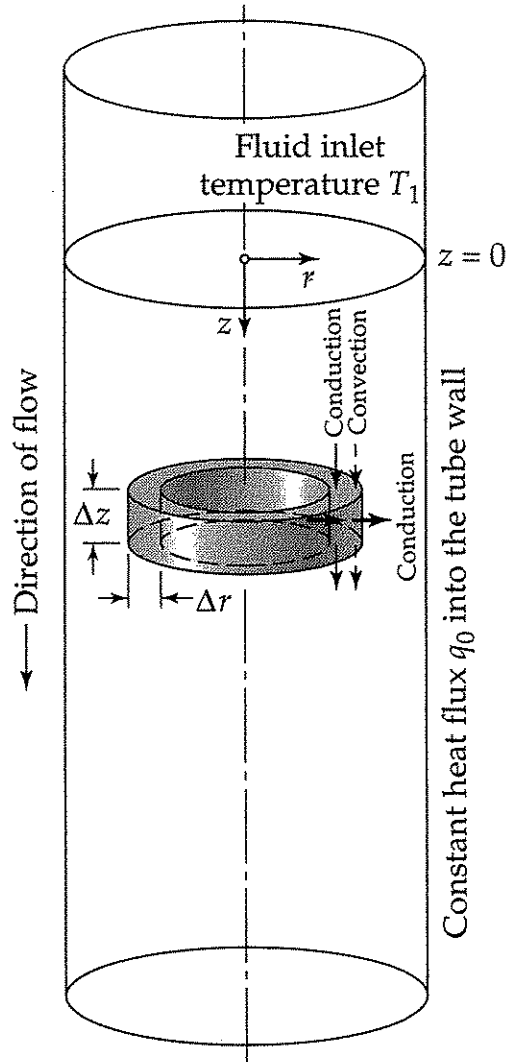
$= 2v_0 \left[1 - \left(\frac{r}{R} \right)^2 \right]$

• The above velocity distribution $v_z(r)$ can be used in the energy balance when the velocity vector $\vec{v} \hat{z}$

$\vec{v} = \hat{e}_r v_r + \hat{e}_\theta v_\theta + \hat{e}_z v_z \hat{=} \hat{e}_z v_z(r, \theta, z) = \hat{e}_z v_z(r)$

$v_z(r, \theta, z) = v_z(r)$

Nonisothermal Fluid Transport with a laminar velocity profile - 4



• Energy flux vector = $\vec{e} = \vec{s}_r e_r + \vec{s}_\theta e_\theta + \vec{s}_z e_z$
 when $\vec{e} = \underbrace{\left(\frac{1}{2}\rho v^2 + \rho \hat{H}\right) \vec{v}}_{\text{convective}} + \underbrace{[\vec{\tau} \cdot \vec{v}]}_{\text{work}} + \underbrace{\vec{q}}_{\text{conduction}}$

• Energy Balance

$$\left(\begin{array}{l} \text{Rate of energy input} \\ \text{by convection transport} \\ \text{\& molecular transport} \end{array} \right) - \left(\begin{array}{l} \text{Rate of energy output} \\ \text{by convective transport} \\ \text{\& molecular transport} \end{array} \right)$$

$$+ \left(\begin{array}{l} \text{Rate of work done} \\ \text{on system by} \\ \text{molecular transport} \end{array} \right) - \left(\begin{array}{l} \text{Rate of work done} \\ \text{by system by} \\ \text{molecular transport} \end{array} \right)$$

$$+ \left(\begin{array}{l} \text{Rate of work done} \\ \text{on system by external} \\ \text{forces} \end{array} \right) + \left(\begin{array}{l} \text{Rate of energy} \\ \text{production} \end{array} \right) = 0 \quad \text{At Steady State}$$

- Total energy in at r
- Total energy out at r
- Total energy in at z
- Total energy out at z+Δz

$$e_r|_r \cdot \lambda \pi r \Delta z = (\lambda \pi r e_r|_r) \Delta z$$

$$e_r|_{r+\Delta r} \cdot \lambda \pi (r+\Delta r) \Delta z = (\lambda \pi r e_r|_{r+\Delta r}) \Delta z$$

$$e_z|_z \cdot \lambda \pi r \Delta r$$

$$e_z|_{z+\Delta z} \cdot \lambda \pi r \Delta r$$

Nonisothermal Fluid Transport with a Lamellar velocity Profile - 5

- work done on fluid by gravity force

$$\Delta N = \rho g \cdot \frac{m}{s^2}$$

$$\rho g_z (2\pi r \Delta r \Delta z) v_z$$

$$\underbrace{\rho \frac{\rho g}{m^3} g_z \frac{m}{s^2}}_{N/m^3} \underbrace{(2\pi r \Delta r \Delta z)}_{m^3} \underbrace{v_z}_{m/s}$$

(Force/volume) (fluid volume) (fluid velocity)

$$\underbrace{2\pi r e_r|_r - 2\pi (r e_r)|_{r+\Delta r}}_{\text{net energy in r-dirn}} + \underbrace{2\pi r \Delta r (e_z)|_z - 2\pi r \Delta r (e_z)|_{z+\Delta z}}_{\text{net energy in z-dirn}} + \underbrace{\rho g_z (2\pi r \Delta r \Delta z) v_z}_{\text{work done on fluid element}}$$

- Divide each term by the differential fluid volume element $2\pi r \Delta r \Delta z$
 & take the limit as $\Delta r \& \Delta z \rightarrow 0$

$$\lim_{\Delta r \rightarrow 0} \frac{2\pi (r e_r)|_r - 2\pi (r e_r)|_{r+\Delta r}}{2\pi r \Delta r \Delta z} + \lim_{\Delta z \rightarrow 0} \frac{2\pi r \Delta r (e_z)|_z - 2\pi r \Delta r (e_z)|_{z+\Delta z}}{2\pi r \Delta r \Delta z} + \frac{\rho g_z (2\pi r \Delta r \Delta z) v_z}{2\pi r \Delta r \Delta z} = 0$$

Cancel out $2\pi \Delta z$ (cancel) $2\pi r \Delta r$

Nonisothermal Fluid Transport with a Lamellar Velocity Profile - 6

- After canceling out like terms, the following eqn is obtained

$$-\lim_{\Delta r \rightarrow 0} \left\{ \frac{(r e_r)|_{r+\Delta r} - (r e_r)|_r}{r \Delta r} \right\} - \lim_{\Delta z \rightarrow 0} \left\{ \frac{(e_z)|_{z+\Delta z} - (e_z)|_z}{\Delta z} \right\} + \rho g_z v_z(r) = 0$$

- Applying the definition for the partial derivatives gives

$$-\frac{1}{r} \frac{\partial}{\partial r} (r e_r) - \frac{\partial e_z}{\partial z} + \rho g_z v_z(r) = 0$$

- Next, write out the components of the energy flux vector

$$\begin{aligned} \vec{e} &= \hat{\delta}_r e_r + \hat{\delta}_\theta e_\theta + \hat{\delta}_z e_z = \hat{\delta}_r e_r + \hat{\delta}_z e_z \text{ since } e_\theta = 0 \\ &= \left(\frac{1}{2} \rho v^2 + \hat{p} \right) \vec{v} + [\overline{\overline{\tau}} \cdot \vec{v}] + \vec{q} \end{aligned}$$

- To evaluate $[\overline{\overline{\tau}} \cdot \vec{v}]$ (dot product of the stress tensor $\overline{\overline{\tau}}$ & velocity vector \vec{v}), it can be shown that

$$[\overline{\overline{\tau}} \cdot \vec{v}] = \sum_i \hat{\delta}_i \left\{ \sum_j \tau_{ij} v_j \right\} \text{ where } i, j \text{ are the coordinate directions}$$

Nonisothermal Fluid Transport with a laminar velocity Profile - 7

- In cylindrical coordinates:

$$\begin{aligned} \left[\overline{\overline{\overline{\nabla \cdot \vec{v}}}} \right] &= \hat{\delta}_r \left\{ \nabla_{rr} v_r + \nabla_{r\theta} v_\theta + \nabla_{rz} v_z \right\} \\ &+ \hat{\delta}_\theta \left\{ \nabla_{\theta r} v_r + \nabla_{\theta\theta} v_\theta + \nabla_{\theta z} v_z \right\} \\ &+ \hat{\delta}_z \left\{ \nabla_{zr} v_r + \nabla_{z\theta} v_\theta + \nabla_{zz} v_z \right\} \end{aligned}$$

$$\left[\overline{\overline{\overline{\nabla \cdot \vec{v}}}} \right]_r = \hat{\delta}_r \left\{ \nabla_{rr} v_r + \nabla_{r\theta} v_\theta + \nabla_{rz} v_z \right\} \quad ; \text{ r-component}$$

$$\left[\overline{\overline{\overline{\nabla \cdot \vec{v}}}} \right]_\theta = \hat{\delta}_\theta \left\{ \nabla_{\theta r} v_r + \nabla_{\theta\theta} v_\theta + \nabla_{\theta z} v_z \right\} \quad ; \text{ } \theta\text{-component}$$

$$\left[\overline{\overline{\overline{\nabla \cdot \vec{v}}}} \right]_z = \hat{\delta}_z \left\{ \nabla_{zr} v_r + \nabla_{z\theta} v_\theta + \nabla_{zz} v_z \right\} \quad ; \text{ z-component}$$

- since $v_r = v_\theta = 0$ & $v_z = v_z(r)$,

$$\left[\overline{\overline{\overline{\nabla \cdot \vec{v}}}} \right]_r = \hat{\delta}_r \nabla_{rz} v_z \quad ; \quad \left[\overline{\overline{\overline{\nabla \cdot \vec{v}}}} \right]_\theta = \hat{\delta}_\theta \nabla_{\theta z} v_z \quad ; \quad \left[\overline{\overline{\overline{\nabla \cdot \vec{v}}}} \right]_z = \hat{\delta}_z \nabla_{zz} v_z$$

Nonisothermal Fluid Transport with a laminar velocity profile - 8

Components of the Fluid Stress Tensor $\overline{\tau}$ for
Newtonian Fluids in Cartesian Coordinates (x, y, z)

$$[\tau = -\mu(\nabla\mathbf{v} + (\nabla\mathbf{v})^t) + (\frac{2}{3}\mu - \kappa)(\nabla \cdot \mathbf{v})\delta]$$

Cartesian coordinates (x, y, z):

$$\tau_{xx} = -\mu \left[2 \frac{\partial v_x}{\partial x} \right] + (\frac{2}{3}\mu - \kappa)(\nabla \cdot \mathbf{v}) \quad (\text{B.1-1})^a$$

$$\tau_{yy} = -\mu \left[2 \frac{\partial v_y}{\partial y} \right] + (\frac{2}{3}\mu - \kappa)(\nabla \cdot \mathbf{v}) \quad (\text{B.1-2})^a$$

$$\tau_{zz} = -\mu \left[2 \frac{\partial v_z}{\partial z} \right] + (\frac{2}{3}\mu - \kappa)(\nabla \cdot \mathbf{v}) \quad (\text{B.1-3})^a$$

$$\tau_{xy} = \tau_{yx} = -\mu \left[\frac{\partial v_y}{\partial x} + \frac{\partial v_x}{\partial y} \right] \quad (\text{B.1-4})$$

$$\tau_{yz} = \tau_{zy} = -\mu \left[\frac{\partial v_z}{\partial y} + \frac{\partial v_y}{\partial z} \right] \quad (\text{B.1-5})$$

$$\tau_{zx} = \tau_{xz} = -\mu \left[\frac{\partial v_x}{\partial z} + \frac{\partial v_z}{\partial x} \right] \quad (\text{B.1-6})$$

in which

$$(\nabla \cdot \mathbf{v}) = \frac{\partial v_x}{\partial x} + \frac{\partial v_y}{\partial y} + \frac{\partial v_z}{\partial z} \quad (\text{B.1-7})$$

^a When the fluid is assumed to have constant density, the term containing $(\nabla \cdot \mathbf{v})$ may be omitted. For monatomic gases at low density, the dilatational viscosity κ is zero.

Nonisothermal Fluid Transport with a Lamellar velocity Profile - 9

Components of the Fluid Stress Tensor $\bar{\tau}$ for

CYLINDRICAL (r, θ, z)

Cylindrical coordinates (r, θ, z) :

$$\begin{aligned}\tau_{rr} &= -\mu \left[2 \frac{\partial v_r}{\partial r} \right] + \left(\frac{2}{3}\mu - \kappa \right) (\nabla \cdot \mathbf{v}) \\ \tau_{\theta\theta} &= -\mu \left[2 \left(\frac{1}{r} \frac{\partial v_\theta}{\partial \theta} + \frac{v_r}{r} \right) \right] + \left(\frac{2}{3}\mu - \kappa \right) (\nabla \cdot \mathbf{v}) \\ \tau_{zz} &= -\mu \left[2 \frac{\partial v_z}{\partial z} \right] + \left(\frac{2}{3}\mu - \kappa \right) (\nabla \cdot \mathbf{v}) \\ \tau_{r\theta} = \tau_{\theta r} &= -\mu \left[r \frac{\partial}{\partial r} \left(\frac{v_\theta}{r} \right) + \frac{1}{r} \frac{\partial v_r}{\partial \theta} \right] \\ \tau_{\theta z} = \tau_{z\theta} &= -\mu \left[\frac{1}{r} \frac{\partial v_z}{\partial \theta} + \frac{\partial v_\theta}{\partial z} \right] \\ \tau_{zr} = \tau_{rz} &= -\mu \left[\frac{\partial v_r}{\partial z} + \frac{\partial v_z}{\partial r} \right]\end{aligned}$$

in which

$$(\nabla \cdot \mathbf{v}) = \frac{1}{r} \frac{\partial}{\partial r} (rv_r) + \frac{1}{r} \frac{\partial v_\theta}{\partial \theta} + \frac{\partial v_z}{\partial z}$$

SPHERICAL (r, θ, ϕ)

Spherical coordinates (r, θ, ϕ) :

$$\begin{aligned}\tau_{rr} &= -\mu \left[2 \frac{\partial v_r}{\partial r} \right] + \left(\frac{2}{3}\mu - \kappa \right) (\nabla \cdot \mathbf{v}) \\ \tau_{\theta\theta} &= -\mu \left[2 \left(\frac{1}{r} \frac{\partial v_\theta}{\partial \theta} + \frac{v_r}{r} \right) \right] + \left(\frac{2}{3}\mu - \kappa \right) (\nabla \cdot \mathbf{v}) \\ \tau_{\phi\phi} &= -\mu \left[2 \left(\frac{1}{r \sin \theta} \frac{\partial v_\phi}{\partial \phi} + \frac{v_r + v_\theta \cot \theta}{r} \right) \right] + \left(\frac{2}{3}\mu - \kappa \right) (\nabla \cdot \mathbf{v}) \\ \tau_{r\theta} = \tau_{\theta r} &= -\mu \left[r \frac{\partial}{\partial r} \left(\frac{v_\theta}{r} \right) + \frac{1}{r} \frac{\partial v_r}{\partial \theta} \right] \\ \tau_{\theta\phi} = \tau_{\phi\theta} &= -\mu \left[\frac{\sin \theta}{r} \frac{\partial}{\partial \theta} \left(\frac{v_\phi}{\sin \theta} \right) + \frac{1}{r \sin \theta} \frac{\partial v_\theta}{\partial \phi} \right] \\ \tau_{\phi r} = \tau_{r\phi} &= -\mu \left[\frac{1}{r \sin \theta} \frac{\partial v_r}{\partial \phi} + r \frac{\partial}{\partial r} \left(\frac{v_\phi}{r} \right) \right]\end{aligned}$$

in which

$$(\nabla \cdot \mathbf{v}) = \frac{1}{r^2} \frac{\partial}{\partial r} (r^2 v_r) + \frac{1}{r \sin \theta} \frac{\partial}{\partial \theta} (v_\theta \sin \theta) + \frac{1}{r \sin \theta} \frac{\partial v_\phi}{\partial \phi}$$

• The term $\nabla \cdot \bar{\mathbf{v}} = 0$ for constant density systems

Nonisothermal Fluid Transport with a Lamellar Velocity Profile - 10

Fourier's Law of Heat Conduction in Various Coordinates

$$\vec{q} = -k \nabla T$$

Cartesian coordinates (x, y, z):

$$q_x = -k \frac{\partial T}{\partial x}$$

$$q_y = -k \frac{\partial T}{\partial y}$$

$$q_z = -k \frac{\partial T}{\partial z}$$

Cylindrical coordinates (r, θ , z):

$$q_r = -k \frac{\partial T}{\partial r}$$

$$q_\theta = -k \frac{1}{r} \frac{\partial T}{\partial \theta}$$

$$q_z = -k \frac{\partial T}{\partial z}$$

Spherical coordinates (r, θ , ϕ):

$$q_r = -k \frac{\partial T}{\partial r}$$

$$q_\theta = -k \frac{1}{r} \frac{\partial T}{\partial \theta}$$

$$q_\phi = -k \frac{1}{r \sin \theta} \frac{\partial T}{\partial \phi}$$

k = thermal conductivity of fluid or solid, W/m·K

^a For mixtures, the term $\sum_\alpha (\bar{H}_\alpha / M_\alpha) j_\alpha$ must be added to $-k \nabla T$ (see Eq. 19.3-3).

Nonisothermal Fluid Transport with a Lamellar Velocity Profile - 11

• Return to the energy balance equation

$$-\frac{1}{r} \frac{\partial}{\partial r} (r e_r) - \frac{\partial e_z}{\partial z} + \rho g_z v_z(r) = 0$$

• Energy flux vector: $\vec{e} = \left(\frac{1}{2} \rho v^2 + p \hat{\pi} \right) \vec{v} + [\vec{\pi} \cdot \vec{v}] + \vec{q}$
 $= \hat{\delta}_r e_r + \hat{\delta}_\theta e_\theta + \hat{\delta}_z e_z$
 $\vec{v} = \hat{\delta}_r v_r + \hat{\delta}_\theta v_\theta + \hat{\delta}_z v_z = \hat{\delta}_z v_z \quad \left\{ \begin{array}{l} \text{since} \\ v_r = v_\theta = 0 \end{array} \right.$

• Components of $\vec{e} = [e_r \ e_\theta \ e_z]^T$; determine one at-a-time

$$e_r = \left(\frac{1}{2} \rho v^2 + p \hat{\pi} \right) v_r + \underbrace{[\vec{\pi} \cdot \vec{v}]_r}_{\uparrow_{r2} v_z} + \underbrace{q_r}_{-k \frac{\partial T}{\partial r}}; \quad v_r = 0 \ \& \ [\vec{\pi} \cdot \vec{v}]_r = \uparrow_{r2} v_z$$

$$\uparrow_{r2} = \uparrow_{2r} = -\mu \left[\frac{\partial v_r}{\partial z} + \frac{\partial v_z}{\partial r} \right] = -\mu \frac{dv_z}{dr} \quad \text{since } v_r = 0; \text{ substitute into above expression}$$

$$e_r = -\left(\mu \frac{dv_z}{dr} \right) v_z - k \frac{\partial T}{\partial r}$$

} final expression for e_r

Nonisothermal Fluid Transport at constant wall Temperature - 12

- We also need $e_z \Rightarrow z$ -component of \vec{e}

$$e_z = \left(\frac{1}{2} \rho v_z^2 + \rho \hat{H} \right) v_z + [\overline{\tau \cdot \vec{v}}]_z + q_z$$

$$v^2 = v_r^2 + v_\theta^2 + v_z^2 = v_z^2 \text{ since } v_r = v_\theta = 0$$

$$[\overline{\tau \cdot \vec{v}}]_z = \tau_{zz} v_z; \tau_{zz} = -\mu \left[2 \frac{\partial v_z}{\partial z} \right] + \left(\frac{2}{3} \mu - \kappa \right) \nabla_i \vec{v} \quad \left\{ \begin{array}{l} \text{From page 9} \\ \text{of notes} \end{array} \right.$$

- Since $v_z = v_z(r)$, then $\partial v_z / \partial z = 0$ & $\nabla_i \vec{v} = 0$ for $\rho = \text{constant}$

This means that $[\overline{\tau \cdot \vec{v}}]_z \equiv 0$;

- $q_z = -k \frac{\partial T}{\partial z}$ { from Fourier's Law of conduction, page 10 of notes }

$$e_z = \left(\frac{1}{2} \rho v_z^2 + \rho \hat{H} \right) v_z + \tau_{zz} v_z + q_z$$

$$= \left(\frac{1}{2} \rho v_z^2 \right) v_z + \rho \hat{H} v_z - \left(2\mu \frac{\partial v_z}{\partial z} \right) v_z - k \frac{\partial T}{\partial z}; \quad \frac{\partial v_z}{\partial z} \equiv 0 \text{ since } v_z = v_z(r)$$

$$e_z = \left(\frac{1}{2} \rho v_z^2 \right) v_z + \rho \hat{H} v_z - k \frac{\partial T}{\partial z}$$

final expression
for e_z

Nonisothermal Fluid Transport with a Laminar Velocity Profile - 13

• Since: $-\frac{1}{r} \frac{\partial}{\partial r}(r e_r) - \frac{\partial e_z}{\partial z} + \rho g_z v_z(r) = 0$

$$e_r = -\left(\mu \frac{dv_z}{dr}\right) v_z - k \frac{\partial T}{\partial r}; \quad e_z = \left(\frac{1}{2} \rho v_z^2\right) v_z + \rho \hat{H} v_z - k \frac{\partial T}{\partial z}$$

• Term 1: $-\frac{1}{r} \frac{\partial}{\partial r}(r e_r) = \frac{1}{r} \frac{\partial}{\partial r} \left[\mu r \frac{dv_z}{dr} v_z + k r \frac{\partial T}{\partial r} \right]$

$$= \frac{\mu}{r} \left[r \left(\frac{dv_z}{dr} \right)^2 \right] + v_z \frac{\mu}{r} \frac{\partial}{\partial r} \left(r \frac{dv_z}{dr} \right) + \frac{k}{r} \frac{\partial}{\partial r} \left(r \frac{\partial T}{\partial r} \right)$$

Term 2: $-\frac{\partial e_z}{\partial z} = -\frac{\partial}{\partial z} \left[\left(\frac{1}{2} \rho v_z^2\right) v_z + \rho \hat{H} v_z - k \frac{\partial T}{\partial z} \right]$

$$= -\frac{1}{2} \rho v_z^2 \left(\frac{\partial v_z}{\partial z} \right) - \rho v_z^2 \frac{\partial v_z}{\partial z} - \rho \frac{\partial}{\partial z} (\hat{H} v_z) + k \frac{\partial^2 T}{\partial z^2}$$

$\swarrow \rightarrow 0$ $\swarrow \rightarrow 0$

$$= -\rho v_z \frac{\partial \hat{H}}{\partial z} - \rho \hat{H} \frac{\partial v_z}{\partial z} + k \frac{\partial^2 T}{\partial z^2}; \quad \frac{\partial v_z}{\partial z} = 0 \text{ since } v_z = v_z(r)$$

$$= -\rho v_z \frac{\partial \hat{H}}{\partial z} + k \frac{\partial^2 T}{\partial z^2}$$

Nonisothermal Fluid Transport at constant wall Temperature - 14

• we showed earlier that if $\bar{H} = \bar{H}(T, r)$, that $d\bar{H} = \bar{c}_p dT$

This means that $\frac{\partial \bar{H}}{\partial z} = \bar{c}_p \frac{\partial T}{\partial z}$ if $\bar{c}_p = \text{constant}$

• It follows that: $-\frac{\partial e_z}{\partial z} = -\rho v_z \bar{c}_p \frac{\partial T}{\partial z} + k \frac{\partial^2 T}{\partial z^2}$

• $-\frac{1}{r} \frac{\partial}{\partial r} (r e_r) - \frac{\partial e_z}{\partial z} + \rho g_z v_z = 0$ { substitute the above expressions

$$\frac{\mu}{r} \left[r \left(\frac{dv_z}{dr} \right)^2 \right] + v_z \frac{\mu}{r} \frac{\partial}{\partial r} \left(r \frac{\partial v_z}{\partial r} \right) + \frac{k}{r} \frac{\partial}{\partial r} \left(r \frac{\partial T}{\partial r} \right) - \rho v_z \bar{c}_p \frac{\partial T}{\partial z} + k \frac{\partial^2 T}{\partial z^2}$$

• Rearrange to: $+ \rho g_z v_z(r) = 0$

$$\rho \bar{c}_p v_z \frac{\partial T}{\partial z} = k \left[\frac{1}{r} \frac{\partial}{\partial r} \left(r \frac{\partial T}{\partial r} \right) + \frac{\partial^2 T}{\partial z^2} \right] + \mu \left(\frac{dv_z}{dr} \right)^2 + v_z \left[\frac{\mu}{r} \frac{\partial}{\partial r} \left(r \frac{\partial v_z}{\partial r} \right) + \rho g_z \right]$$

where $v_z = v_z(r) = 2v_0 \left[1 - \left(\frac{r}{R} \right)^2 \right]$